## EE 4CL4 – Control System Design

## Solutions to Homework Assignment #6

- 1. For the system with the open-loop frequency response generating the Nyquist plot shown in Fig. 1, estimate the:
  - a. stability gain margin,
  - b. stability phase margin, and
  - c. sensitivity peak.



a. The stability gain margin  $M_g \stackrel{\Delta}{=} -20 \log_{10}(|a|)$ . From Fig. 1,  $a \approx -0.77 \Rightarrow M_g \approx 2.27$  dB.

- b. The stability phase margin  $M_f \stackrel{\scriptscriptstyle \Delta}{=} \phi$ . From Fig. 1,  $\phi \approx 0.135 \text{ rad} \Rightarrow M_f \approx 0.135 \text{ rad or } 7.73^\circ$ .
- c. The sensitivity peak  $\max_{\omega} S_o(j\omega) = 1/\eta$ . From Fig. 1,  $\eta \approx 0.12 \Rightarrow \max_{\omega} S_o(j\omega) \approx 8.33$ .

(25 pts)

## 2. The nominal model for a plant is given by:

$$G_o(s) = \frac{1}{(s+1)(-s+2)}.$$

Assume that this plant has to be controlled in a one-d.o.f. feedback loop such that the closedloop characteristic polynomial is dominated by the factor  $s^2 + 7s + 25$ . Using the pole placement method, choose an appropriate minimum degree  $A_{cl}(s)$  and synthesize a *biproper* controller C(s) that has forced integration (i.e., one pole at s = 0). (25 pts)

We first notice that a minimum degree biproper controller (with integration) requires  $A_{cl}(s)$  of degree 4 (= 2n). We thus choose  $A_{cl}(s) = (s^2 + 7s + 25)(s + 10)^2$ . The choice of the double pole at s = -10 is arbitrary but for the requirement that they should generate modes faster than those produced by the factor  $s^2 + 7s + 25$ .

The associated Diophantine equation is:

$$(s^{2} - s - 2)s(l_{1}s + l_{0}) + (-1)(p_{2}s^{2} + p_{1}s + p_{0}) = (s^{2} + 7s + 25)(s + 10)^{2},$$

producing the pole-assignment matrix equation:

$$\begin{bmatrix} 1 & 0 & 0 & 0 & 0 \\ -1 & 1 & 0 & 0 & 0 \\ -2 & -1 & -1 & 0 & 0 \\ 0 & -2 & 0 & -1 & 0 \\ 0 & 0 & 0 & 0 & -1 \end{bmatrix} \begin{bmatrix} l_1 \\ l_0 \\ p_2 \\ p_1 \\ p_0 \end{bmatrix} = \begin{bmatrix} 1 \\ 27 \\ 265 \\ 1200 \\ 2500 \end{bmatrix} \Rightarrow \begin{bmatrix} l_1 \\ l_0 \\ p_2 \\ p_1 \\ p_0 \end{bmatrix} = \begin{bmatrix} 1 \\ 28 \\ -295 \\ -1256 \\ -2500 \end{bmatrix},$$

We thus obtain:

$$C(s) = \frac{-(295s^2 + 1256s + 2500)}{s(s+28)}$$

3. Consider the feedback control of an unstable plant. Prove that the controller output u(t), exhibits undershoot for any step reference and for any step-output disturbance. (25 pts)

The transfer function from the reference and the output disturbance to the controller output is given by:

$$U(s) = S_{uo}(s)(R(s) - D_o(s))$$

When the plant has unstable poles, they cannot be cancelled and thus they appear as non-minimumphase zeros in  $S_{uo}(s)$ . If  $z_o > 0$  is any non-minimum-phase zero in  $S_{uo}(s)$ , then  $S_{uo}(z_o) = 0$ . From Lemma 4.1 on Page 81 of Goodwin et al.:

$$\int_{0}^{\infty} u(t) e^{-z_0 t} dt = \lim_{s \to z_0} U(s) = 0.$$

For this equation to be satisfied, u(t) will necessarily exhibit undershoot (i.e., be negative for some period of the response) for any step reference and step-output disturbance.

4. The nominal model for a plant is given by:

$$G_o(s) = \frac{5(s-1)}{(s+1)(s-5)}$$

This plant has to be controlled in a one-d.o.f. unity-feedback loop.

- a. Determine the time-domain *integral constraints* for the plant input u(t), the plant output y(t), and the controller error e(t) in the loop. Assume exact inversion at  $\omega = 0$  (see page 210 of Goodwin et al.) and step-like reference and disturbances (input and output).
- b. Discuss why the control of this nominal plant especially difficult. Hint: What constraints should be placed on the closed-loop bandwidth? (25 pts)
- a. In this particular case we have that the plant model and the controller satisfies  $B_o(1) = 0$ ;  $A_o(5) = 0$ ; L(0) = 0. The zero in L(s) at s = 0 is required for exact inversion at  $\omega = 0$ .

The constraints for the sensitivities derive from the interpolation constraints required to achieve internal stability (no cancellation of unstable poles and NMP zeros). These constraints are:

$$S_o(1) = 1; S_o(5) = 0; T_o(1) = 0; T_o(5) = 1; S_{io}(1) = 0; S_{uo}(5) = 0.$$

First, the reference effect will be seen in y(t), u(t) and e(t), and their integral properties:

$$Y(s) = T_o(s)\frac{1}{s} \Rightarrow \int_0^\infty y(t)e^{-t}dt = 0$$
  
$$\Rightarrow \int_0^\infty y(t)e^{-5t}dt = \frac{1}{5}$$
  
$$U(s) = S_{uo}(s)\frac{1}{s} \Rightarrow \int_0^\infty u(t)e^{-5t}dt = 0$$
  
$$E(s) = S_o(s)\frac{1}{s} \Rightarrow \int_0^\infty e(t)e^{-5t}dt = 0$$
  
$$\Rightarrow \int_0^\infty e(t)e^{-t}dt = 1$$

Likewise, the effect of a unit step *input* disturbance is:

$$Y(s) = S_{io}(s)\frac{1}{s} \Longrightarrow \int_0^\infty y(t)e^{-t}dt = 0$$
$$U(s) = -T_o(s)\frac{1}{s} \Longrightarrow \int_0^\infty u(t)e^{-t}dt = 0$$
$$\Longrightarrow \int_0^\infty u(t)e^{-5t}dt = -\frac{1}{5}$$
$$E(s) = -S_{io}(s)\frac{1}{s} \Longrightarrow \int_0^\infty e(t)e^{-t}dt = 0$$

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The effect of a unit step *output* disturbance is:

$$Y(s) = S_o(s)\frac{1}{s} \Longrightarrow \int_0^\infty y(t)e^{-5t}dt = 0$$
  
$$\Rightarrow \int_0^\infty y(t)e^{-t}dt = 1$$
  
$$U(s) = -S_{uo}(s)\frac{1}{s} \Longrightarrow \int_0^\infty u(t)e^{-5t}dt = 0$$
  
$$E(s) = -S_o(s)\frac{1}{s} \Longrightarrow \int_0^\infty e(t)e^{-5t}dt = 0$$
  
$$\Rightarrow \int_0^\infty e(t)e^{-t}dt = -1$$

- b. This case is especially difficult because of contradicting requirements:
  - The open-loop NMP zero at s = 1 sets an *upper bound* for the closed-loop bandwidth, since the integral constraint  $\int_0^\infty y(t)e^{-t}dt = 0$  derived above says that a plant output which settles much faster than  $e^{-t}$  will exhibit a *large undershoot*.
  - The unstable open-loop pole at s = 5 sets a *lower bound* for the closed loop bandwidth, since the integral constraint  $\int_0^\infty e(t)e^{-5t}dt = 0$  derived above says that a plant output which settles much slower than  $e^{-5t}$  will exhibit a *large overshoot*.