

Tim Davidson

Instinct-base Design

Models

Feedback systems

Control System Design

EE3CL4: Introduction to Linear Control Systems Post-Reading-Week Conceptual Review

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Outline

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EE 3CL4, PBWCB

Instinct-based Design

Models

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2 Models

3 Feedback systems



Informal Review

- So what have we done so far?
- Instinct-based design

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- Proportional control of walking to the half-way line; worked quite well, if a bit slow
- Proportional control of drone hover; did not work so well
- Weaknesses and lack of reliability suggested model-based design

Models

Instinct-base

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• Simplified models for mechanical systems

- Free body diagrams; akin to node/mesh analysis
- Force = mass x acceleration
- If model is linear and does not change in time;

 —> linear time-invariant (LTI) differential equations
- Analysis can be simplified using Laplace Transforms
 - Can learn a lot about system behaviour from poles and zeros
 - Also simplifies analysis of interconnections of systems (block diagram models)

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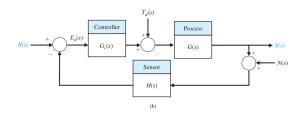
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Control of LTI systems



- *G*(*s*): system to be controlled
- H(s): chosen sensor We will focus on "good" sensors that can be approximated by $H(s) \approx 1$
- $G_c(s)$: controller that we will design

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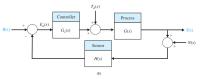
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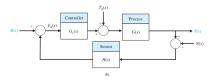


With
$$H(s) = 1$$
 and $E(s) = R(s) - Y(s)$,
 $E(s) = \frac{1}{1 + G_c(s)G(s)} R(s) - \frac{G(s)}{1 + G_c(s)G(s)} T_d(s) + \frac{G_c(s)G(s)}{1 + G_c(s)G(s)} N(s)$

Many properties of these closed-loop transfer functions depend strongly on the open-loop transfer function $G_c(s)G(s)$.

- for good tracking, want $G_c(s)G(s)$ to be large when R(s) is dominant
- for good disturbance rejection, want G_c(s)G(s) to be large when T_d(s) is significant
- for good noise suppression, want G_c(s)G(s) to be small when N(s) is dominant

Control of LTI Systems



With H(s) = 1 and E(s) = R(s) - Y(s),

$$E(s) = rac{1}{1+G_c(s)G(s)} \ R(s) - rac{G(s)}{1+G_c(s)G(s)} \ T_d(s) + rac{G_c(s)G(s)}{1+G_c(s)G(s)} \ N(s)$$

- Also, the steady-state error due to step and ramp inputs
 - depends strongly on the number of integrators in the open loop
 - when finite (and not zero) depends on open-loop pole and zero positions

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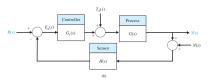
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With H(s) = 1 and E(s) = R(s) - Y(s),

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- Transient input-output properties depend quite strongly on closed-loop pole and zero positions
 - For second-order systems with no zeros, we can quantify relationships between closed-loop pole positions and settling time, and between closed-loop pole positions and overshoot/damping



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Design of LTI Control Systems

- How do we start to quantify our insights?
- We can use the Routh-Hurwitz technique to determine choices of the controller parameters that lead to a stable closed loop
- We can also do this for settling time, but that is algebraically quite complicated
- We can handle steady-state error constraints with simple equations
- We combined these ideas for a two-parameter design approach with stability, steady-state error and settling time constraints
- Enabled us to bound the areas in the design parameter space that gave us the desired performance.
- Extension to three controller parameters makes visualization more difficult; extension to four parameters really difficult.



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Design of LTI Control Systems

- Looks like we might need a more flexible design technique
- So many things depend on closed-loop pole (and zero) positions,
 - stability, transient response (including settling time, damping), steady-state errors due to step and ramp,
- let's try to get an idea of the path that the closed-loop poles take as we change one of our design parameters.
- Let's also try to get an idea about how to choose the other design parameters so that that path goes were we would like it to go.
- Finally, let's try to get an idea of how to choose the value of the design parameter so that we are able to place the closed-loop poles at specified points on the path