

Tim Davidson

Transfer  
functions

Frequency  
Response

Plotting the  
freq. resp.

Mapping  
Contours

Nyquist's  
criterion

Ex: servo, P control  
Ex: unst., P control  
Ex: unst., PD contr.  
Ex: RHP Z, P contr.

Nyquist's  
Stability  
Criterion as a  
Design Tool

Relative Stability  
Gain margin and  
Phase margin  
Relationship to  
transient response

# EE3CL4:

## Introduction to Linear Control Systems

### Section 8: Frequency Domain Techniques

Tim Davidson

McMaster University

Winter 2020

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# Outline

- 1 Transfer functions
- 2 Frequency Response
- 3 Plotting the freq. resp.
- 4 Mapping Contours
- 5 Nyquist's criterion
  - Ex: servo, P control
  - Ex: unst., P control
  - Ex: unst., PD contr.
  - Ex: RHP Z, P contr.
- 6 Nyquist's Stability Criterion as a Design Tool
  - Relative Stability
  - Gain margin and Phase margin
  - Relationship to transient response

# Transfer Functions: A Quick Review

- Consider a transfer function

$$G(s) = K \frac{\prod_i (s + z_i)}{\prod_j (s + p_j)}$$

- Zeros:  $-z_i$ ; Poles:  $-p_j$
- Note that  $s + z_i = s - (-z_i)$ ,
- This is the vector from  $-z_i$  to  $s$
- Magnitude:

$$|G(s)| = |K| \frac{\prod_i |s + z_i|}{\prod_j |s + p_j|} = |K| \frac{\text{prod. dist's from zeros to } s}{\text{prod. dist's from poles to } s}$$

- Phase:

$$\angle G(s) = \angle K + \text{sum angles from zeros to } s \\ - \text{sum angles from poles to } s$$

# Frequency Response

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- For a stable, linear, time-invariant (LTI) system, the steady state response to a sinusoidal input is a sinusoid of the same frequency but possibly different magnitude and different phase
- Sinusoids are the eigenfunctions of convolution
- If input is  $A \cos(\omega_0 t + \theta)$  and steady-state output is  $B \cos(\omega_0 t + \phi)$ , then the complex number  $B/A e^{j(\phi - \theta)}$  is called the frequency response of the system at frequency  $\omega_0$ .

# Frequency Response, II

- If a stable LTI system has a transfer function  $G(s)$ , then the frequency response at  $\omega_0$  is  $G(s)|_{s=j\omega_0}$
- What if the system is unstable?

## Plotting the frequency response

Transfer functions

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Nyquist's Stability Criterion as a Design Tool

Relative Stability  
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- For each  $\omega$ ,  $G(j\omega)$  is a complex number.
- How should we plot it?
- $G(j\omega) = |G(j\omega)|e^{j\angle G(j\omega)}$   
Plot  $|G(j\omega)|$  versus  $\omega$ , and  $\angle G(j\omega)$  versus  $\omega$
- Plot  $20 \log_{10}(|G(j\omega)|)$  versus  $\log_{10}(\omega)$ , and  $\angle G(j\omega)$  versus  $\log_{10}(\omega)$
- $G(j\omega) = \text{Re}(G(j\omega)) + j \text{Im}(G(j\omega))$   
Plot the curve  $(\text{Re}(G(j\omega)), \text{Im}(G(j\omega)))$  on an “x–y” plot  
Equiv. to curve  $|G(j\omega)|e^{j\angle G(j\omega)}$  as  $\omega$  changes (polar plot)

# Polar plot, example 1

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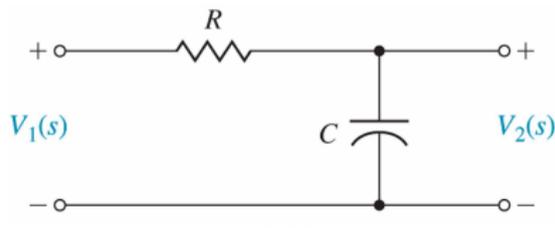
Nyquist's Stability Criterion as a Design Tool

Relative Stability

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Let's consider the example of an RC circuit



- $G(s) = \frac{V_2(s)}{V_1(s)} = \frac{1}{1+sRC}$
- $G(j\omega) = \frac{1}{1+j\omega/\omega_1}$ , where  $\omega_1 = 1/(RC)$ .
- $G(j\omega) = \frac{1}{1+(\omega/\omega_1)^2} - j \frac{\omega/\omega_1}{1+(\omega/\omega_1)^2}$
- $G(j\omega) = \frac{1}{\sqrt{1+(\omega/\omega_1)^2}} e^{-j \operatorname{atan}(\omega/\omega_1)}$

# Polar plot, example 1

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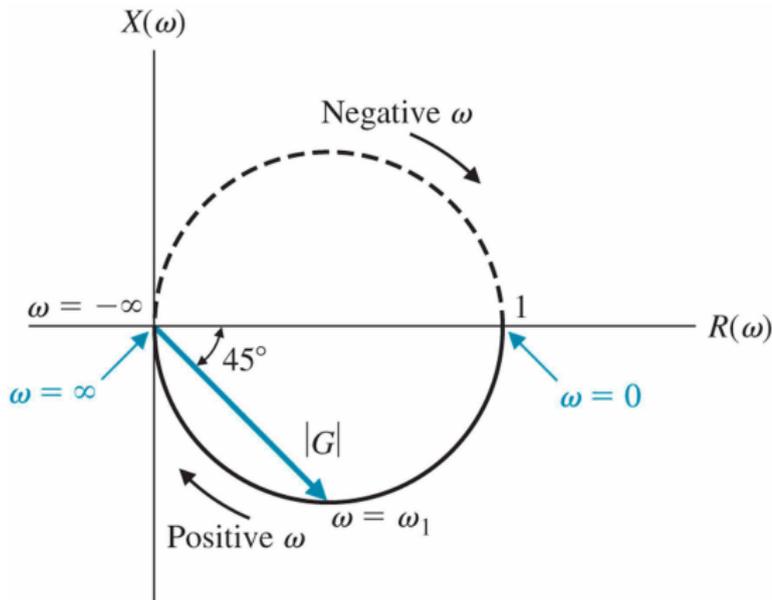
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Nyquist's Stability Criterion as a Design Tool

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- $G(j\omega) = \frac{1}{1+(\omega/\omega_1)^2} - j\frac{\omega/\omega_1}{1+(\omega/\omega_1)^2}$
- $G(j\omega) = \frac{1}{\sqrt{1+(\omega/\omega_1)^2}} e^{-j\text{atan}(\omega/\omega_1)}$



## Polar plot, example 2

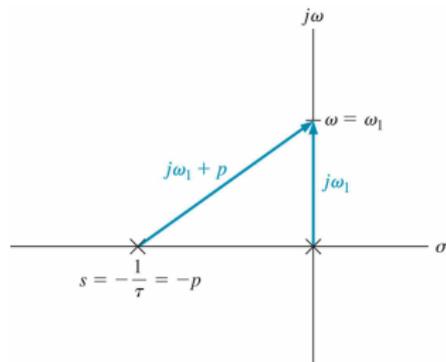
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transient responseConsider  $G(s) = \frac{K}{s(s\tau+1)}$ .

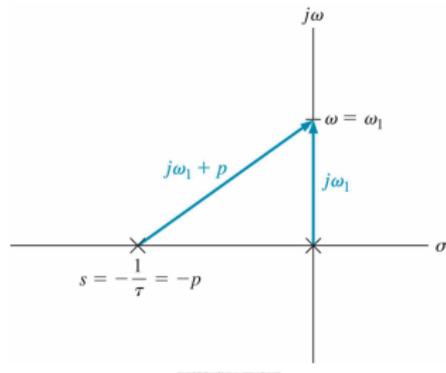
- Poles at origin and  $s = -1/\tau$ .
- To use geometric insight to plot polar plot,

$$\text{rewrite as } G(s) = \frac{K/\tau}{s(s+1/\tau)}$$

- Then  $|G(j\omega)| = \frac{K/\tau}{|j\omega||j\omega+1/\tau|}$   
and  $\angle G(j\omega) = -\angle(j\omega) - \angle(j\omega + 1/\tau)$

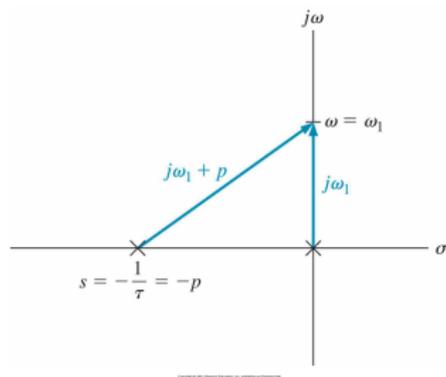


$$\text{Polar plot, ex. 2, } G(s) = \frac{K/\tau}{s(s+1/\tau)}$$



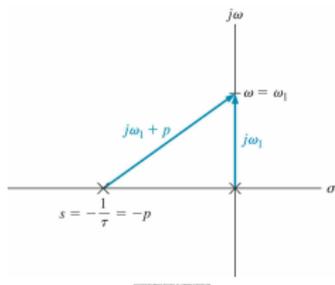
- When  $\omega \rightarrow 0^+$ ,  $|G(j\omega)| \rightarrow \infty$ ,  $\angle G(j\omega) \rightarrow -90^\circ$  from below  
Tricky
- To get a better feel, write  $G(j\omega) = \frac{-K\omega^2\tau}{\omega^2 + \omega^4\tau^2} - j\frac{\omega K}{\omega^2 + \omega^4\tau^2}$   
Hence, as  $\omega \rightarrow 0^+$ ,  $G(j\omega) \rightarrow -K\tau - j\infty$
- As  $\omega$  increases, distances from poles to  $j\omega$  increase.  
Hence  $|G(j\omega)|$  decreases
- As  $\omega$  increases, angle from pole at  $-1/\tau$  increases.  
Hence  $\angle G(j\omega)$  becomes more negative

# Polar plot, ex. 2, $G(s) = \frac{K/\tau}{s(s+1/\tau)}$



- When  $\omega = 1/\tau$ ,  $G(j\omega) = (K/\tau)/((1/\tau)(\sqrt{2}/\tau)) e^{-j(90^\circ + 45^\circ)}$   
i.e.,  $G(j\omega)|_{\omega=1/\tau} = (K\tau/\sqrt{2})e^{-j135^\circ}$
- As  $\omega$  approaches  $+\infty$ , both distances from poles get large.  
Hence  $|G(j\omega)| \rightarrow 0$
- As  $\omega$  approaches  $+\infty$ , angle from  $-1/\tau$  approaches  $-90^\circ$   
from below. Hence  $\angle G(j\omega)$  approaches  $-180^\circ$  from below

$$\text{Polar plot, ex. 2, } G(s) = \frac{K/\tau}{s(s+1/\tau)}$$

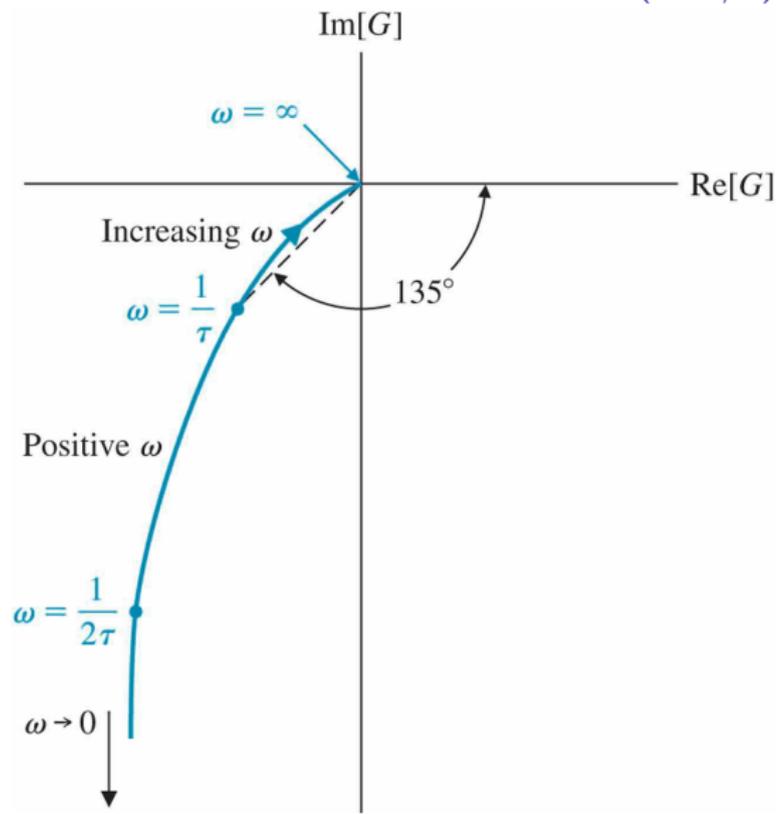


## Summary

- As  $\omega \rightarrow 0^+$ ,  $G(j\omega) \rightarrow -K\tau - j\infty$
- As  $\omega$  increases,  $|G(j\omega)|$  decreases,  $\angle G(j\omega)$  becomes more negative
- When  $\omega = 1/\tau$ ,  $G(j\omega) = (K/\sqrt{2})e^{-j135^\circ}$
- As  $\omega$  approaches  $+\infty$ ,  $G(j\omega)$  approaches zero from angle  $-180^\circ$

- Transfer functions
- Frequency Response
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- Nyquist's criterion
  - Ex: servo, P control
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- Nyquist's Stability Criterion as a Design Tool
  - Relative Stability
  - Gain margin and Phase margin
  - Relationship to transient response

# Polar plot, ex. 2, $G(s) = \frac{K/\tau}{s(s+1/\tau)}$



# Bode Diagrams

- Bode magnitude plot

$$20 \log_{10} |G(j\omega)| \text{ against } \log_{10} \omega$$

- Bode phase plot

$$\angle G(j\omega) \text{ against } \log_{10} \omega$$

- In 2CJ4 we developed rules to help sketch these plots
- In this course we will use these sketches to design controllers

## Sketching Bode Diagrams

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Nyquist's Stability Criterion as a Design Tool

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- Consider generic transfer function of LTI system

$$G(s) = \frac{K \prod_i (s + z_i) \prod_k (s^2 + 2\zeta_k \omega_{n,k} s + \omega_{n,k}^2)}{s^N \prod_j (s + p_j) \prod_r (s^2 + 2\zeta_{d,k} \omega_{nd,r} s + \omega_{nd,r}^2)}$$

where  $z_i$  and  $p_j$  are real.

- Unfortunately, not in the form that we are used to for Bode diagrams
- Divide numerator by  $\prod_i z_i \prod_k \omega_{n,k}^2$
- Similarly for denominator
- Then if  $\tilde{K} = K \prod_i z_i \prod_k \omega_{n,k}^2 / (\prod_j p_j \prod_r \omega_{nd,r}^2)$ ,

$$G(s) = \frac{\tilde{K} \prod_i (1 + s/z_i) \prod_k (1 + 2\zeta_k (s/\omega_{n,k}) + (s/\omega_{n,k})^2)}{s^N \prod_j (1 + s/p_j) \prod_r (1 + 2\zeta_{d,k} (s/\omega_{nd,r}) + (s/\omega_{nd,r})^2)}$$

## Sketching Bode Diagrams, II

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- Now, frequency response can be written as:

$$G(j\omega) = \frac{\tilde{K} \prod_i (1 + j\omega/z_i)}{(j\omega)^N \prod_j (1 + j\omega/p_j)} \times \frac{\prod_k (1 + 2\zeta_k(j\omega/\omega_{n,k}) + (j\omega/\omega_{n,k})^2)}{\prod_r (1 + 2\zeta_{d,k}(j\omega/\omega_{nd,r}) + (j\omega/\omega_{nd,r})^2)}$$

- Four key components:
  - Gain,  $\tilde{K}$
  - Poles (or zeros) at origin
  - Poles and zeros on real axis
  - Poles and zeros in complex conjugate pairs
- Each contributes to the Bode Diagram

# Bode Magnitude diagram

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$$G(j\omega) = \frac{\tilde{K} \prod_i (1 + j\omega/z_i)}{(j\omega)^N \prod_j (1 + j\omega/p_j)} \times \frac{\prod_k (1 + 2\zeta_k(j\omega/\omega_{n,k}) + (j\omega/\omega_{n,k})^2)}{\prod_r (1 + 2\zeta_{d,k}(j\omega/\omega_{nd,r}) + (j\omega/\omega_{nd,r})^2)}$$

- Bode Magnitude diagram:

$20 \log_{10} |G(j\omega)|$  against  $\log_{10} \omega$

- $20 \log_{10} |G(j\omega)|$  is

Sum of  $20 \log_{10}$  of components of numerator

– sum of  $20 \log_{10}$  of components of denominator

## Components for magnitude

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$$G(j\omega) = \frac{\tilde{K} \prod_i (1 + j\omega/z_i)}{(j\omega)^N \prod_j (1 + j\omega/p_j)} \times \frac{\prod_k (1 + 2\zeta_k(j\omega/\omega_{n,k}) + (j\omega/\omega_{n,k})^2)}{\prod_r (1 + 2\zeta_{d,k}(j\omega/\omega_{nd,r}) + (j\omega/\omega_{nd,r})^2)}$$

- Poles at origin: slope starts at  $-20N$  dB/dec
- Gain  $|\tilde{K}|$  incorporated in position of that sloping line
- First order component in numerator:  
increase slope by 20 dB/dec at  $\omega = z_i$
- First order component in denominator:  
decrease slope by 20 dB/dec at  $\omega = p_j$
- Second order components:  
increase or decrease slope by 40 dB/dec at  $\omega = \omega_n$

# Bode Phase Diagram

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$$G(j\omega) = \frac{\tilde{K} \prod_i (1 + j\omega/z_i)}{(j\omega)^N \prod_j (1 + j\omega/p_j)} \times \frac{\prod_k (1 + 2\zeta_k(j\omega/\omega_{n,k}) + (j\omega/\omega_{n,k})^2)}{\prod_r (1 + 2\zeta_{d,k}(j\omega/\omega_{nd,r}) + (j\omega/\omega_{nd,r})^2)}$$

- Bode Phase Diagram

$\angle G(j\omega)$  against  $\log_{10} \omega$

- $\angle G(j\omega)$  is

Sum of phases of components of numerator

– sum of phases of components of denominator

# Components

$$G(j\omega) = \frac{\tilde{K} \prod_i (1 + j\omega/z_i)}{(j\omega)^N \prod_j (1 + j\omega/p_j)} \times \frac{\prod_k (1 + 2\zeta_k(j\omega/\omega_{n,k}) + (j\omega/\omega_{n,k})^2)}{\prod_r (1 + 2\zeta_{d,k}(j\omega/\omega_{nd,r}) + (j\omega/\omega_{nd,r})^2)}$$

- Phase of  $\tilde{K}$
- Poles at origin:  $-N90^\circ$
- First order component in numerator:  
linear phase change of  $+90^\circ$  over  $\omega \in [z_i/10, 10z_i]$
- First order component in denominator:  
linear phase change of  $-90^\circ$  over  $\omega \in [p_j/10, 10p_j]$
- Second order components:  
phase change of  $\pm 180^\circ$  around  $\omega = \omega_n$

# Graphically

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Mapping Contours

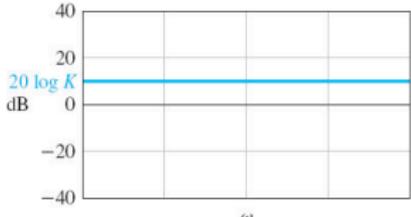
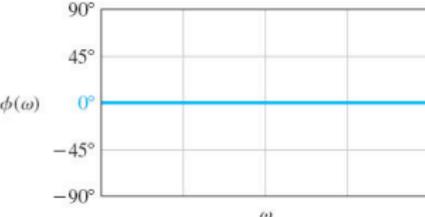
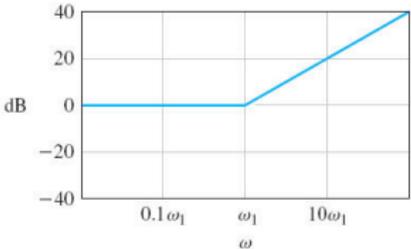
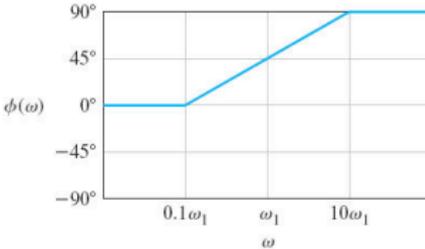
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Nyquist's Stability Criterion as a Design Tool

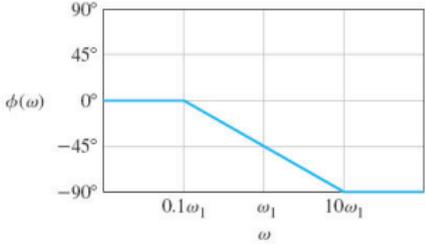
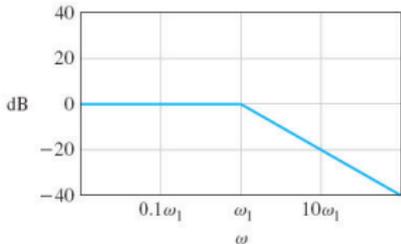
- Relative Stability
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**Table 8.3 Asymptotic Curves for Basic Terms of a Transfer Function**

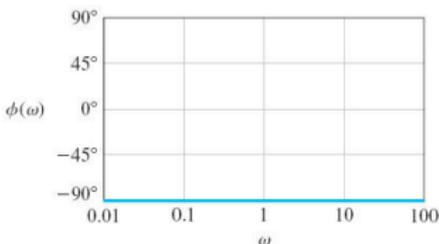
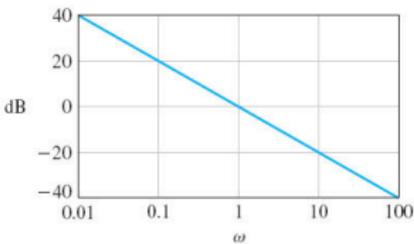
Term	Magnitude $20 \log G $	Phase $\phi(\omega)$
1. Gain, $G(j\omega) = K$		
2. Zero, $G(j\omega) = 1 + j\omega/\omega_1$		

# Graphically

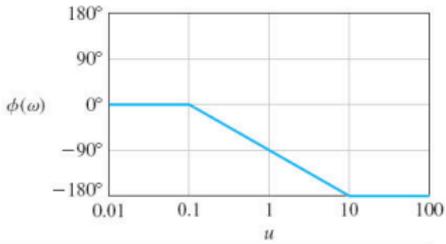
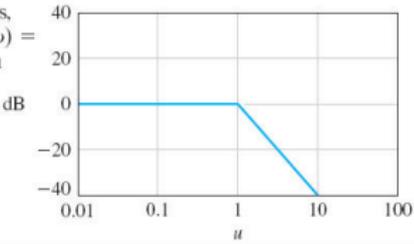
3. Pole,  
 $G(j\omega) = (1 + j\omega/\omega_1)^{-1}$



4. Pole at the origin,  
 $G(j\omega) = 1/j\omega$



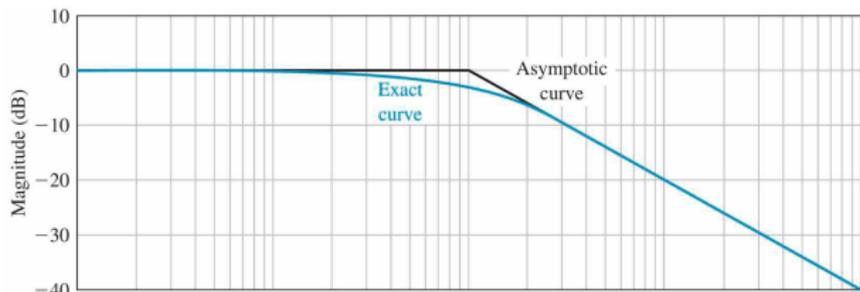
5. Two complex poles,  
 $0.1 < \zeta < 1, G(j\omega) = (1 + j2\zeta u - u^2)^{-1}$   
 $u = \omega/\omega_n$



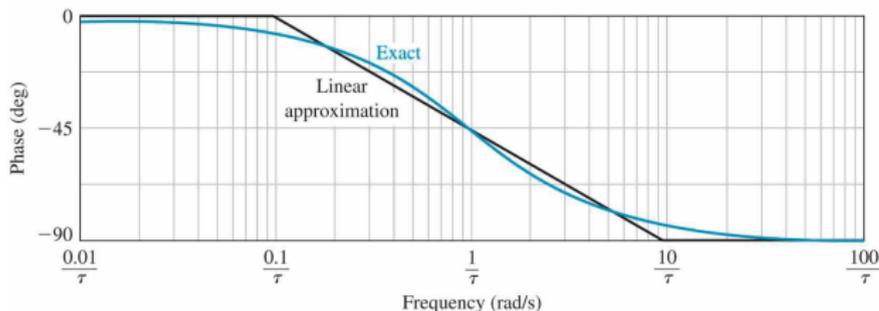
# Accuracy of Bode Sketches

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## Isolated first order pole (analogous for zero)



(a)



(b)

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Nyquist's Stability Criterion as a Design Tool

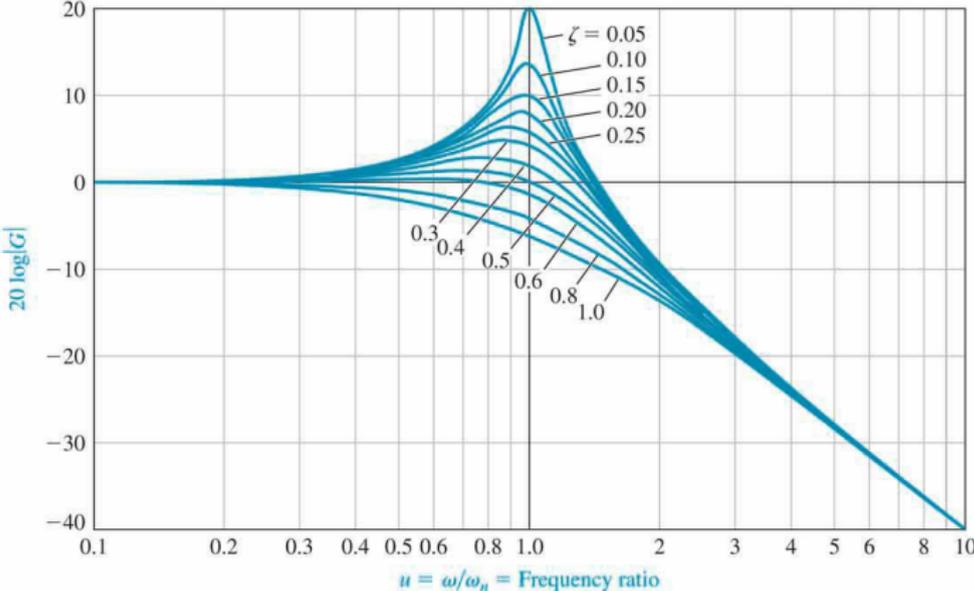
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# Accuracy of Bode Sketches

## Isolated complex conjugate pair of poles



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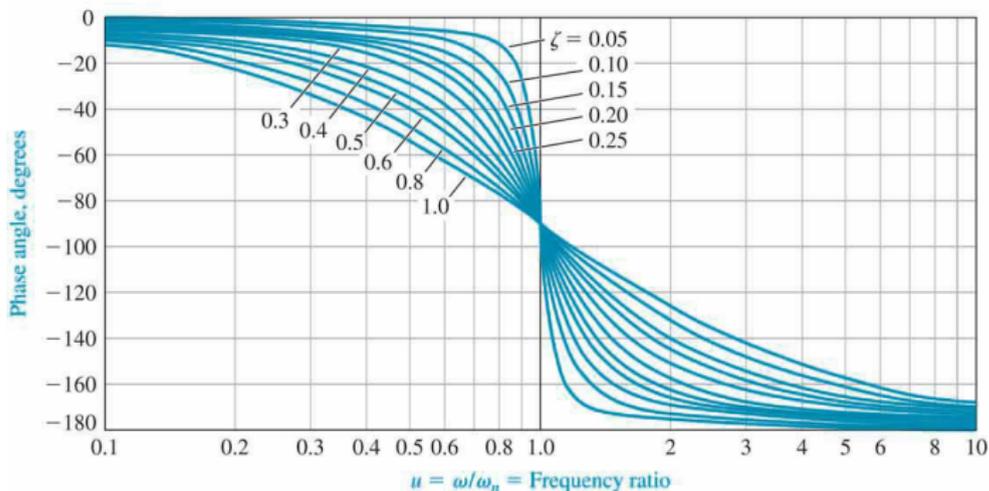
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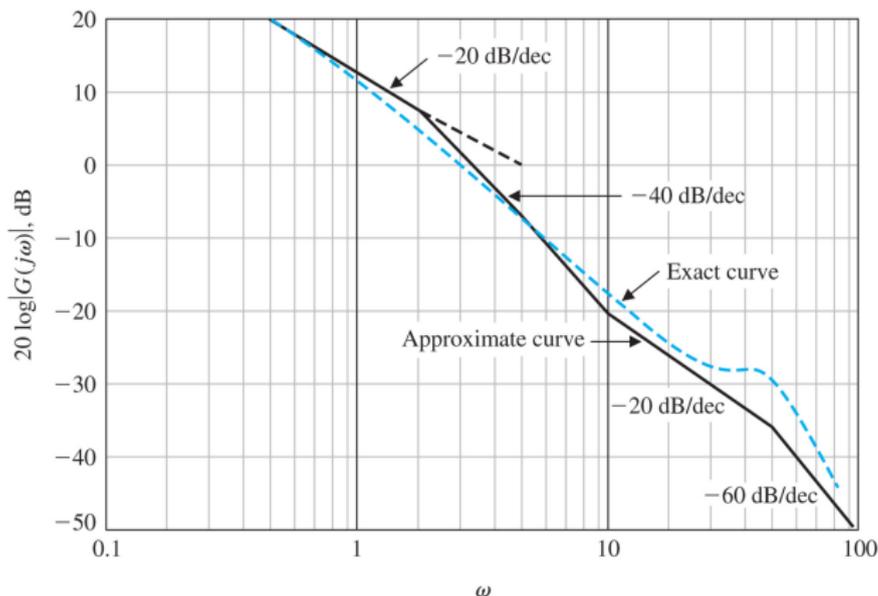
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## Isolated complex conjugate pair of poles



# Example

$$G(j\omega) = \frac{5(1 + j\omega/10)}{j\omega(1 + j\omega/2)(1 + 0.6(j\omega/50) + (j\omega/50)^2)}$$



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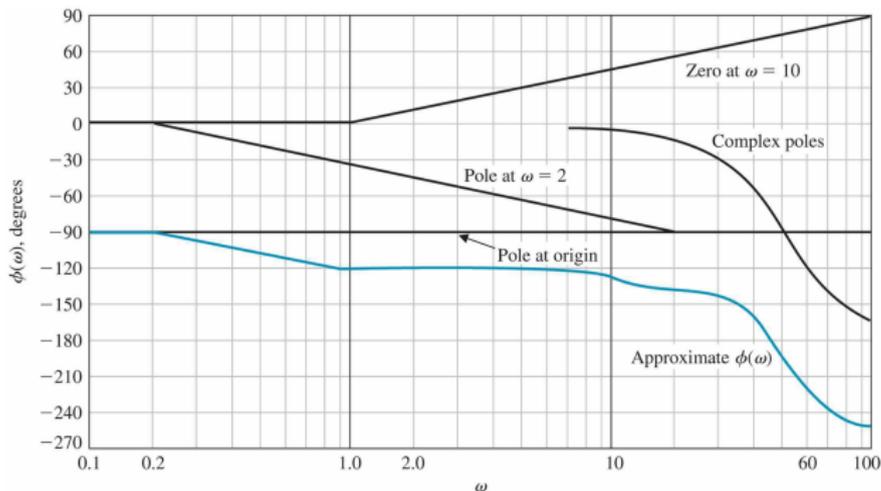
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# Example

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$$G(j\omega) = \frac{5(1 + j\omega/10)}{j\omega(1 + j\omega/2)(1 + 0.6(j\omega/50) + (j\omega/50)^2)}$$



Transfer functions

Frequency Response

Plotting the freq. resp.

Mapping Contours

Nyquist's criterion

Ex: servo, P control  
Ex: unst., P control  
Ex: unst., PD contr.  
Ex: RHP Z, P contr.

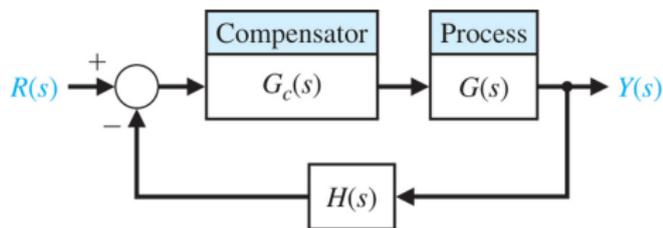
Nyquist's Stability Criterion as a Design Tool

Relative Stability  
Gain margin and Phase margin  
Relationship to transient response

# Introduction

- We have seen techniques that determine stability of a system:
  - Routh-Hurwitz
  - root locus
- However, both of them require a model for the plant
- Today: frequency response techniques
  - Although they work best with a model
  - For an open-loop stable plant, they also work with measurements
- Key result: Nyquist's stability criterion
- Design implications: Bode techniques based on gain margin and phase margin

## Characteristic equation



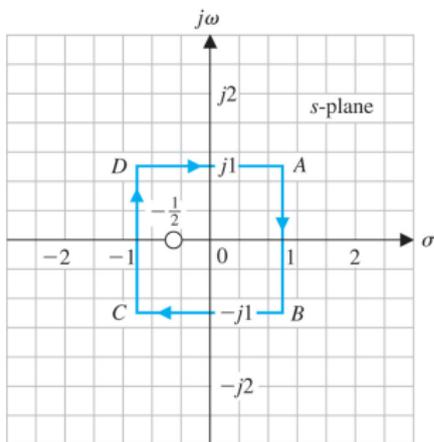
- To determine the stability of the system we need to examine the characteristic equation:

$$F(s) = 1 + L(s) = 0$$

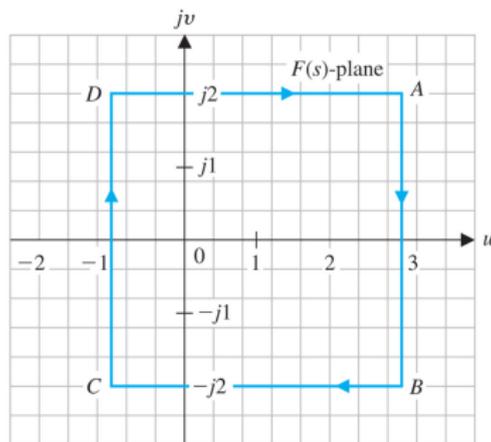
where  $L(s) = G_c(s)G(s)H(s)$ .

- The key result involves mapping a closed contour of values of  $s$  to a closed contour of values of  $F(s)$ .
- We will investigate the idea of mappings first

## Simple example



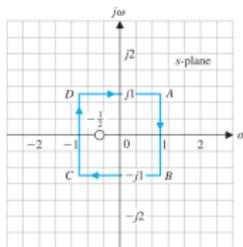
(a)



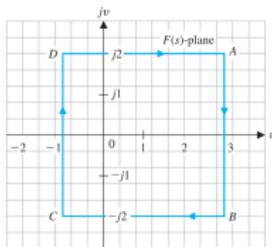
(b)

- Set  $F(s) = 2s + 1$
- Map the square in the "s-plane" to the contour in the "F(s)-plane"

## Area enclosed



(a)



(b)

- How might we define area enclosed by a closed contour?
- We will be perfectly rigorous, but will go against mathematical convention
- Define area enclosed to be that to the right when the contour is traversed clockwise
- What you see when moving clockwise with eyes right
- Sometimes we say that this area is the area “inside” the clockwise contour
- Notions of “enclosed” or “inside” will be applied to contours in the s-plane

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Ex: unst., P control

Ex: unst., PD contr.

Ex: RHP Z, P contr.

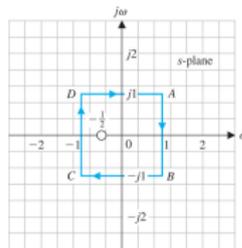
Relative Stability

Gain margin and

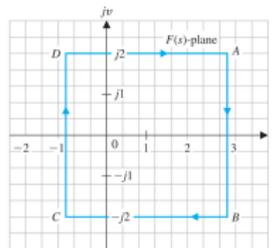
Phase margin

Relationship to transient response

# Encirclement



(a)



(b)

- In the  $F(s)$ -plane, we will be interested in the notion of encirclement of the origin
- A contour is said to encircle the origin in the clockwise direction, if the contour completes a  $360^\circ$  revolution around the origin in the clockwise direction.
- A contour is said to encircle the origin in the anti-clockwise direction, if the contour completes a  $360^\circ$  revolution around the origin in the anti-clockwise direction.
- We will say that an anti-clockwise encirclement is a “negative” clockwise encirclement

## Example with rational $F(s)$

Tim Davidson

Transfer functions

Frequency Response

Plotting the freq. resp.

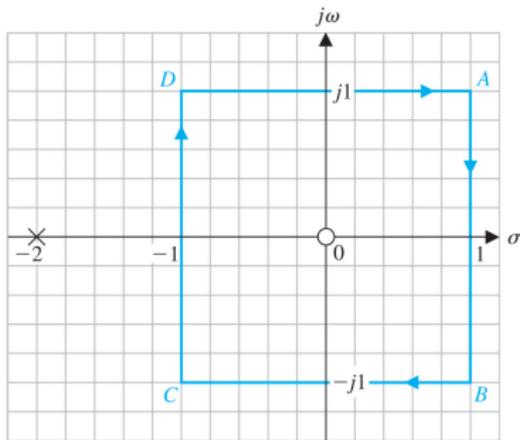
Mapping Contours

Nyquist's criterion

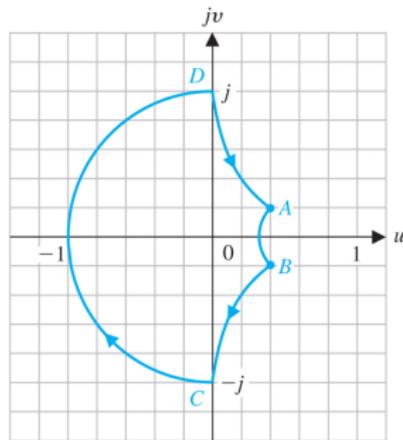
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Nyquist's Stability Criterion as a Design Tool

Relative Stability  
Gain margin and Phase margin  
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(a)



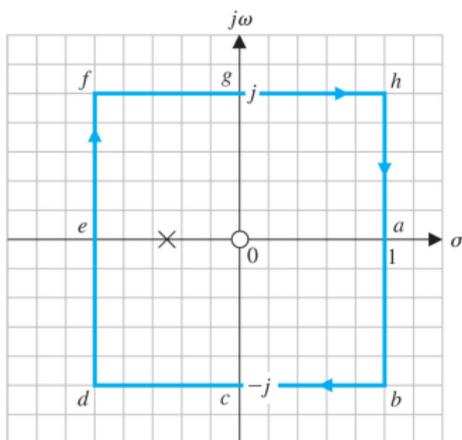
(b)

- A mapping for  $F(s) = \frac{s}{s+2}$
- Note that  $s$ -plane contour encloses the zero of  $F(s)$
- How many times does the  $F(s)$ -plane contour encircle the origin in the clockwise direction?

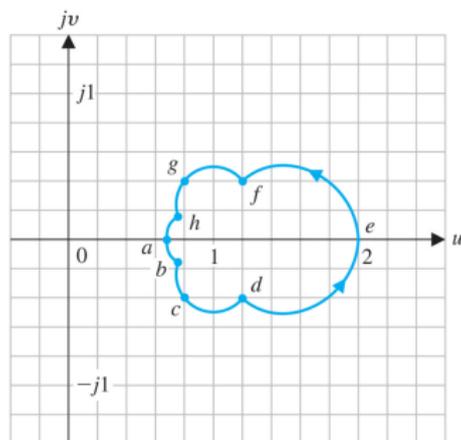
# Cauchy's Theorem

- Nyquist's Criterion is based on Cauchy's Theorem:
  - Consider a rational function  $F(s)$
  - If the clockwise traversal of a contour  $\Gamma_s$  in the  $s$ -plane encloses  $Z$  zeros and  $P$  poles of  $F(s)$  and does not go through any poles or zeros
  - then the corresponding contour in the  $F(s)$ -plane,  $\Gamma_F$  encircles the origin  $N = Z - P$  times in the clockwise direction
- A sketch of the proof later.
- First, some examples

## Example 1



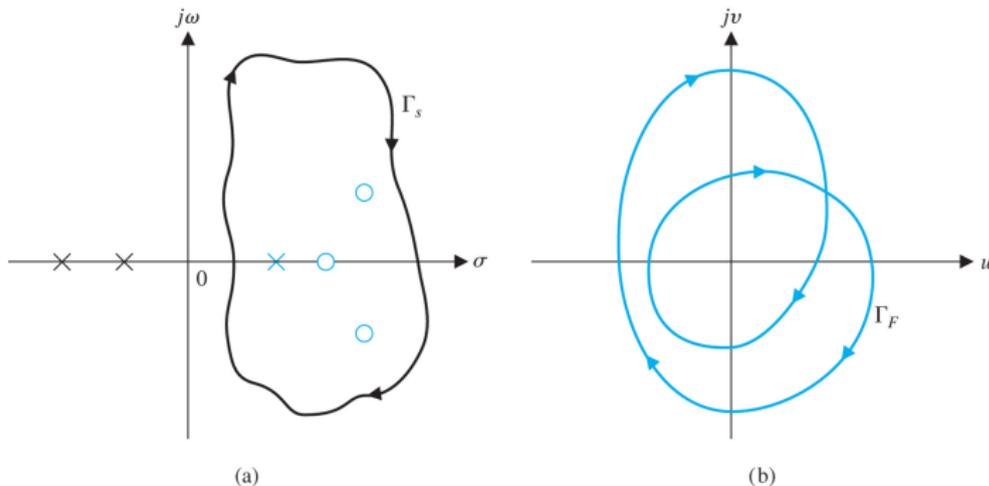
(a)



(b)

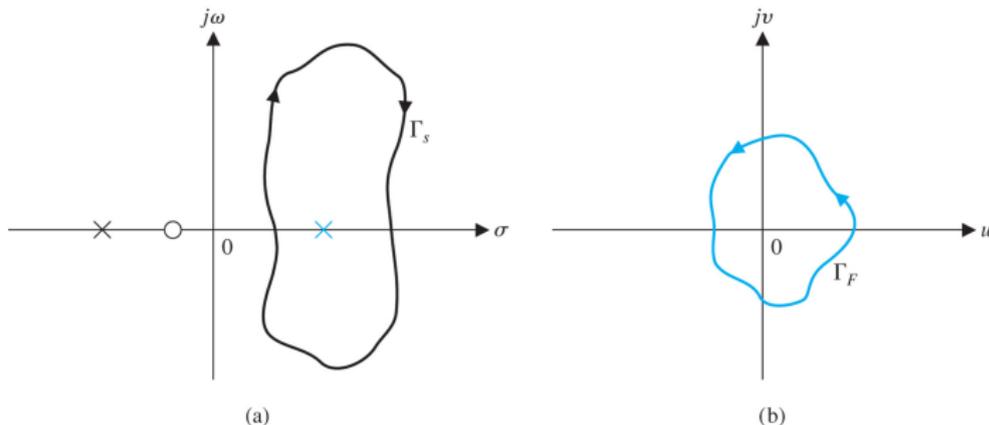
- A mapping for  $F(s) = \frac{s}{s+1/2}$
- s-plane contour encloses a zero and a pole
- Theorem suggests no clockwise encirclements of origin of  $F(s)$ -plane
- This is what we have!

## Example 2



- $s$ -plane contour encloses 3 zeros and a pole
- Theorem suggests 2 clockwise encirclements of the origin of the  $F(s)$ -plane

## Example 3



- $s$ -plane contour encloses one pole
- Theorem suggests -1 clockwise encirclements of the origin of the  $F(s)$ -plane
- That is, one anti-clockwise encirclement

# Informal Justification of Cauchy's Theorem

Tim Davidson

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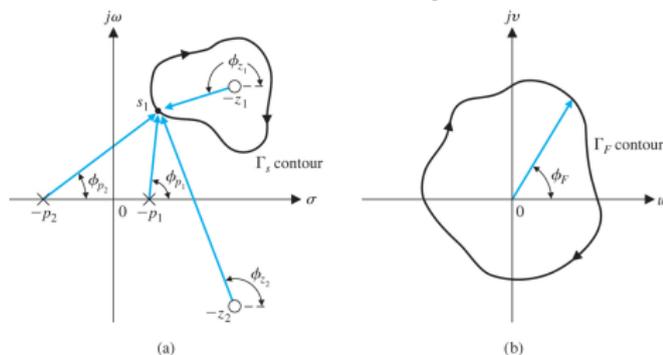
Ex: RHP Z, P contr.

Nyquist's Stability Criterion as a Design Tool

Relative Stability

Gain margin and Phase margin

Relationship to transient response



- Consider the case of  $F(s) = \frac{(s+z_1)(s+z_2)}{(s+p_1)(s+p_2)}$
- $\angle F(s_1) = \phi_{z_1} + \phi_{z_2} - \phi_{p_1} - \phi_{p_2}$
- As the contour is traversed the nett contribution from  $\phi_{z_1}$  is 360 degrees
- As contour is traversed, the nett contribution from other angles is 0 degrees
- Hence, as contour is traversed,  $\angle F(s)$  changes by 360 degrees. One encirclement!

Ex: servo, P control

Ex: unst., P control

Ex: unst., PD contr.

Ex: RHP Z, P contr.

Relative Stability

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Phase margin

Relationship to  
transient response

## Informal Justification

- Extending this to any number of poles and zeros inside the contour
- For a closed contour, the change in  $\angle F(s)$  is  $360Z - 360P$
- Hence  $F(s)$  encircles origin  $Z - P$  times

# Cauchy's Theorem (Review)

Transfer  
functions

Frequency  
Response

Plotting the  
freq. resp.

Mapping  
Contours

Nyquist's  
criterion

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Nyquist's  
Stability  
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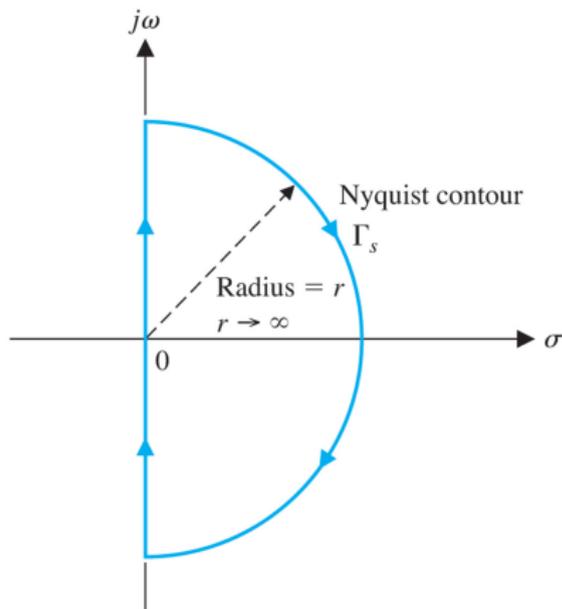
Relationship to  
transient response

- Consider a rational function  $F(s)$
- If the clockwise traversal of a contour  $\Gamma_s$  in the  $s$ -plane encloses  $Z$  zeros and  $P$  poles of  $F(s)$  and does not go through any poles or zeros
- then the corresponding contour in the  $F(s)$ -plane,  $\Gamma_F$  encircles the origin  $N = Z - P$  times in the clockwise direction

## Nyquist's goal

- Nyquist was concerned about testing for stability
- How might one use Cauchy Theorem to examine this?
- Perhaps choose  $F(s) = 1 + L(s)$ , as this determines stability
- Which contour should we use?

## Nyquist's contour



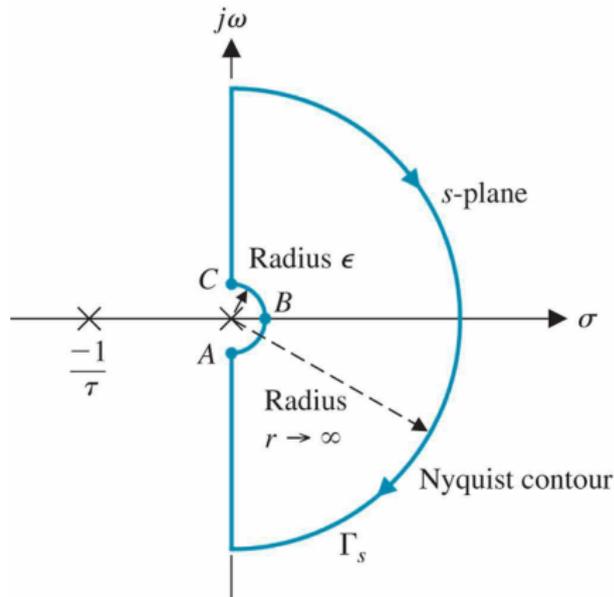
Actually, we have to be careful regarding poles and zeros on the  $j\omega$ -axis, including the origin.

Standard approach is to indent contour so that it goes to the right of any such poles or zeros

# Modified Nyquist contour

Tim Davidson

Here's an example for a model like that of the motor in the lab.



Transfer functions

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Plotting the freq. resp.

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Nyquist's criterion

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Ex: RHP Z, P contr.

Nyquist's Stability Criterion as a Design Tool

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Relationship to transient response

## Coarse Applic. of Cauchy

Transfer functions

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Nyquist's Stability Criterion as a Design Tool

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Phase margin

Relationship to transient response

- Recall that the zeros of  $F(s) = 1 + L(s)$  are the poles of the closed loop
- Let  $P$  denote the number of right half plane poles of  $F(s)$
- The number of right half plane zeros of  $F(s)$  is  $N + P$ , where  $N$  is the number of clockwise encirclements of the origin made by the image of Nyquist's contour in the  $F(s)$  plane.
- A little difficult to parse.
- Perhaps we can apply Cauchy's Theorem in a more sophisticated way.

## Towards Nyquist's Criterion

Transfer functions

Frequency Response

Plotting the freq. resp.

Mapping Contours

Nyquist's criterion

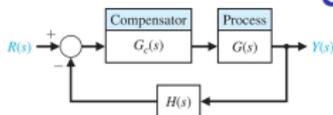
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Nyquist's Stability Criterion as a Design Tool

Relative Stability  
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- $F(s) = 1 + L(s)$ , where  $L(s)$  is the open loop transfer function
- Encirclement of the origin in  $F(s)$ -plane is the same as encirclement of  $-1$  in the  $L(s)$ -plane
- This is more convenient, because  $L(s)$  is often factorized, and hence we can easily determine  $P$
- Now that we are dealing with  $L(s)$ ,  $P$  is the number of right-half plane poles of the open loop transfer function
- If we handle the remainder of the components of Cauchy's theorem carefully we obtain:

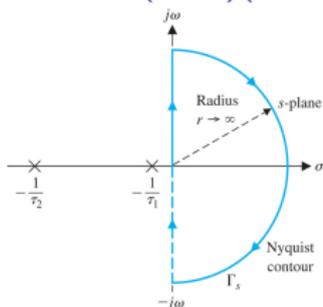
## Nyquist's Criterion: Simplified statement



- Consider a unity feedback system with an open loop transfer function  $L(s) = G_c(s)G(s)H(s)$ , with no z's or p's on  $j\omega$ -axis
- Let  $P_L$  denote the number of poles of  $L(s)$  in RHP
- Consider the Nyquist Contour in the  $s$ -plane
- Let  $\Gamma_L$  denote image of Nyquist Contour under  $L(s)$
- Let  $N_L$  denote the number of clockwise encirclements that  $\Gamma_L$  makes of the point  $(-1, 0)$
- **Nyquist's Stability Criterion:**  
Number of closed-loop poles in RHP  $= N_L + P_L$

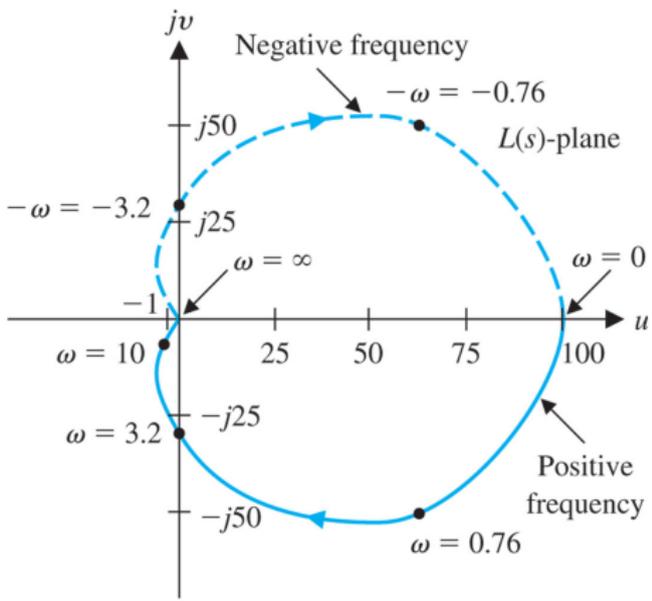
- Note that for a stable open loop, the closed-loop is stable if the image of the Nyquist contour does not encircle  $(-1, 0)$ .

$$\text{Ex: } L(s) = \frac{1000}{(s+1)(s+10)} \text{ (stable)}$$



- For  $0 \leq \omega < \infty$ :
  - No zeros, two poles.
  - $|L(0)| = 1000/(1 \times 10) = 100$ ;  $\angle L(0) = -0 - 0 = 0$
  - Distances from poles to  $j\omega$  is increasing; hence  $|L(j\omega)|$  is decreasing
  - Angles from poles to  $j\omega$  are increasing; hence  $\angle L(j\omega)$  is decreasing
  - As  $\omega \rightarrow \infty$ ,  $|L(j\omega)| \rightarrow 0$ ,  $\angle L(j\omega) \rightarrow -180^\circ$
- Recall that  $L(-j\omega) = L(j\omega)^*$
- Remember to examine the  $r \rightarrow \infty$  part of the curve

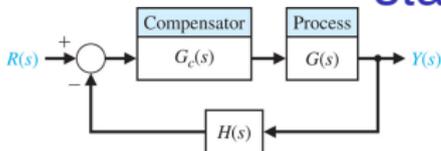
$$\text{Ex: } L(s) = \frac{1000}{(s+1)(s+10)} \text{ (stable)}$$



Note: No encirclements of  $(-1, 0) \implies$  closed loop is stable

- Transfer functions
- Frequency Response
- Plotting the freq. resp.
- Mapping Contours
- Nyquist's criterion**
  - Ex: servo, P control
  - Ex: unst., P control
  - Ex: unst., PD contr.
  - Ex: RHP Z, P contr.
- Nyquist's Stability Criterion as a Design Tool
  - Relative Stability
  - Gain margin and Phase margin
  - Relationship to transient response

## Nyquist's Criterion: Refined statement



- Consider a unity feedback system with an open loop transfer function  $L(s) = G_c(s)G(s)H(s)$ ,
- Let  $P_L$  denote the number of poles of  $L(s)$  in **open** RHP
- Consider the **modified** Nyquist Contour in the  $s$ -plane **looping to the right of any poles or zeros on the  $j\omega$ -axis**
- Let  $\Gamma_L$  denote image of **mod.** Nyquist Contour under  $L(s)$
- Let  $N_L$  denote the number of clockwise encirclements that  $\Gamma_L$  makes of the point  $(-1, 0)$
- **Nyquist's Stability Criterion:**
  - Number of closed-loop poles in **open** RHP =  $N_L + P_L$
- Now we can handle open-loop poles and zeros on  $j\omega$ -axis

## Example: Pole of $L(s)$ at origin

Transfer  
functions

Frequency  
Response

Plotting the  
freq. resp.

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Contours

Nyquist's  
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Nyquist's  
Stability  
Criterion as a  
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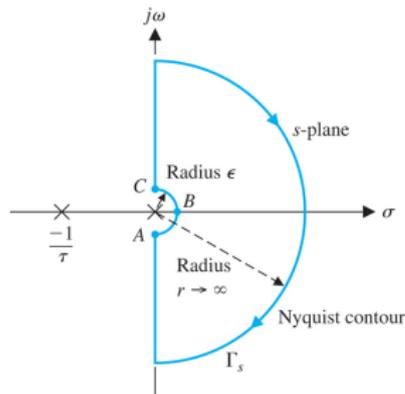
- Consider

$$L(s) = \frac{K}{s(\tau s + 1)}$$

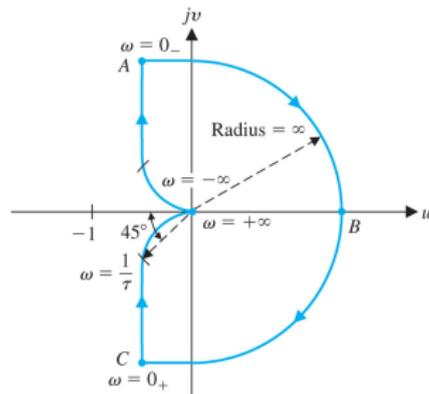
- Like in servomotor
- Problem with the original Nyquist contour
- It goes through a pole!
- Cauchy's Theorem does not apply
- Must modify Nyquist Contour to go around pole
- Then Nyquist Criterion can be applied

## Example: Pole of $L(s)$ at origin

Tim Davidson



(a)



(b)

Transfer functions

Frequency Response

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Ex: servo, P control

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Nyquist's Stability Criterion as a Design Tool

Relative Stability

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Relationship to transient response

Now three key aspects of the curve

- Around the origin
- Positive frequency axis; remember negative freq. axis yields conjugate
- At  $\infty$

Ex: servo, P control

Ex: unst., P control

Ex: unst., PD contr.

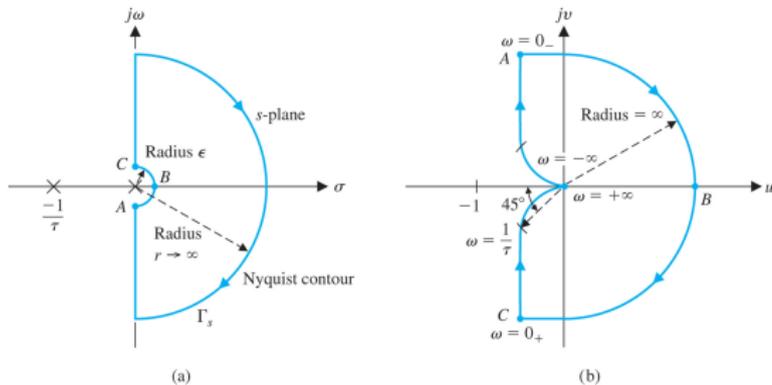
Ex: RHP Z, P contr.

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# Around the origin



- $L(s) = \frac{K}{s(\tau s + 1)}$
- Around the origin,  $s = \epsilon e^{j\phi}$ , where  $\phi$  goes from  $-90^\circ$  to  $90^\circ$
- In the  $L(s)$  plane:  $\lim_{\epsilon \rightarrow 0} L(\epsilon e^{j\phi})$
- This is:  $\lim_{\epsilon \rightarrow 0} \frac{K}{\epsilon e^{j\phi}} = \lim_{\epsilon \rightarrow 0} \frac{K}{\epsilon} e^{-j\phi}$

Ex: servo, P control

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Ex: unst., PD contr.

Ex: RHP Z, P contr.

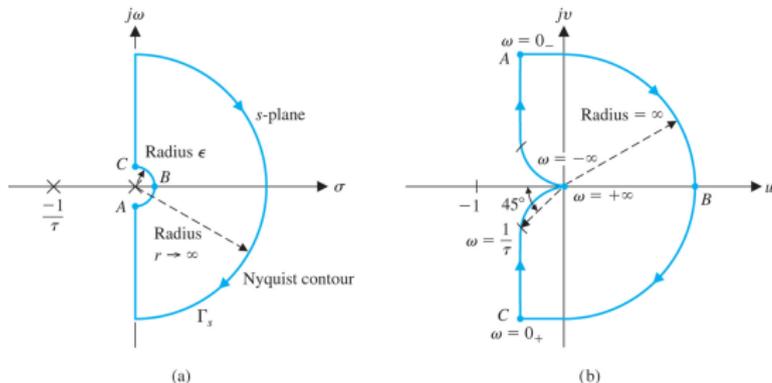
Relative Stability

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## Up positive $j\omega$ -axis



- For  $0 < \omega < \infty$ ,  $L(j\omega) = \frac{K}{\omega\sqrt{1+\omega^2\tau^2}} e^{-j(90^\circ + \text{atan}(\omega\tau))}$
- For small  $\omega$ ,  $L(j\omega)$  is large with phase  $-90^\circ$   
Actually, as we worked out in a previous lecture, as  $\omega \rightarrow 0^+$ ,  $L(j\omega) \rightarrow -K\tau - j\infty$
- For large  $\omega$ ,  $L(j\omega)$  is small with phase  $-180^\circ$
- For  $\omega = 1/\tau$ ,  $L(j\omega) = K\tau/\sqrt{2} e^{-j135^\circ}$

For  $s = re^{j\theta}$  for large  $r$

Tim Davidson

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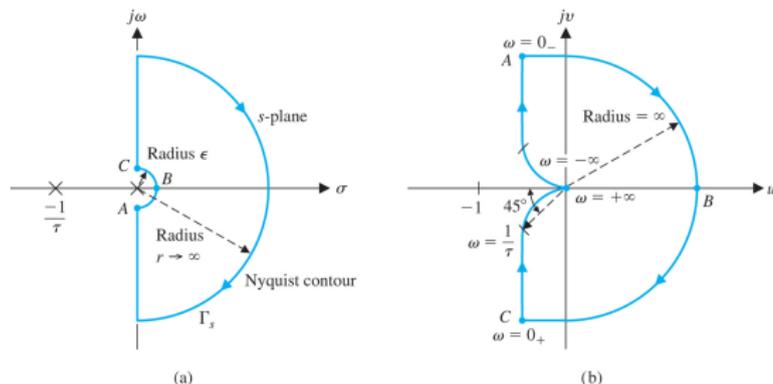
Ex: RHP Z, P contr.

Nyquist's Stability Criterion as a Design Tool

Relative Stability

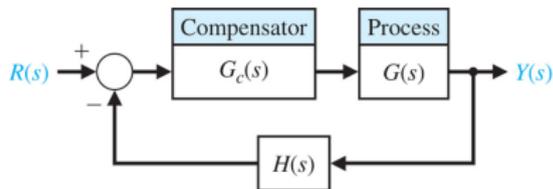
Gain margin and Phase margin

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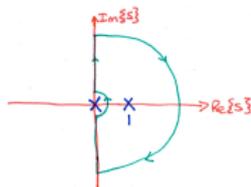
- For  $s = re^{j\theta}$  with large  $r$ , and  $\theta$  from  $+90^\circ$  to  $-90^\circ$ ,
- $\lim_{r \rightarrow \infty} L(re^{j\theta}) = \frac{K}{\tau r^2} e^{-j2\theta}$
- How many encirclements of  $-1$  in  $L(s)$  plane? None
- Implies that closed loop is stable for all positive  $K$
- Consistent with what we know from root locus (Lab. 2)

## Example with open loop RHP pole, proportional control



- Consider  $G(s) = \frac{1}{s(s-1)}$
- Essentially the same as plant model for VTOL aircraft example in root locus section
- Consider prop. control,  $G_c(s) = K_1$ , and  $H(s) = 1$ .
- Hence,  $L(s) = \frac{K_1}{s(s-1)}$
- Observe that  $L(s)$  has a pole in RHP; hence  $P_L = 1$

## Ex. with open loop RHP pole



- $L(s) = \frac{K_1}{s(s-1)}$ . For  $s = j\omega$  and  $0 < \omega < \infty$ ,
 
$$L(j\omega) = \frac{-K_1}{1 + \omega^2} + j \frac{K_1}{\omega(1 + \omega^2)} = \frac{K_1}{\omega\sqrt{1 + \omega^2}} \angle (90^\circ + \text{atan}(\omega))$$
- For  $\omega \rightarrow 0^+$ ,  $L(j\omega) \rightarrow -K_1 + j\infty$
- As  $\omega$  increases, real and imag. parts decrease, imag. part decreases faster
- Equiv. magnitude decreases, phase increases
- For  $\omega \rightarrow \infty$ ,  $L(j\omega)$  is small with angle  $+180^\circ$
- Conjugate for  $-\infty < \omega < 0$
- What about when  $s = \epsilon e^{j\theta}$  for  $-90^\circ \leq \theta \leq 90^\circ$ ?
 
$$L(s) = \frac{K_1}{\epsilon} \angle (-180^\circ - \theta)$$

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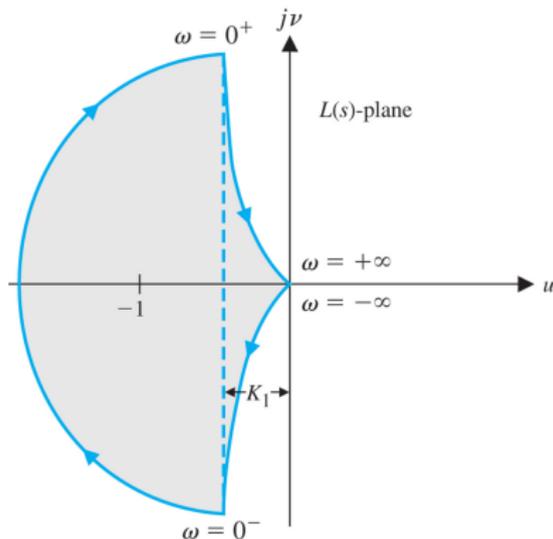
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Relative Stability

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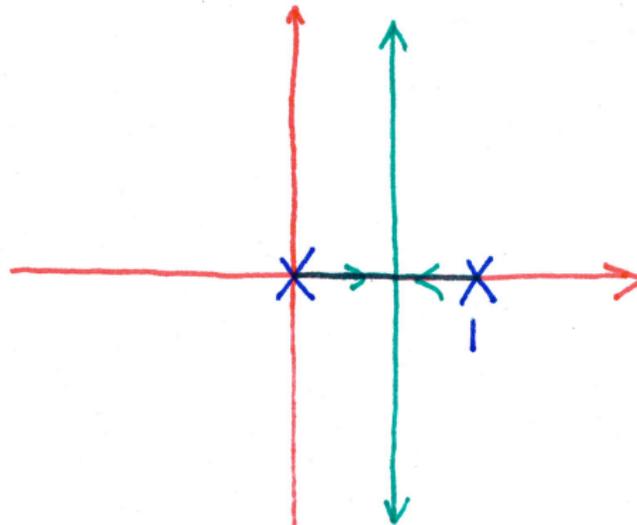
Relationship to transient response

# Example with open loop RHP pole



- Recall  $P_L = 1$
- Number clockwise encirclements of  $(-1, 0)$  is 1
- Hence there are two closed loop poles in the RHP for all positive values of  $K_1$
- Consistent with root locus analysis

$$\text{Root locus of } L(s) = \frac{1}{s(s-1)}$$



Tim Davidson

Transfer functions

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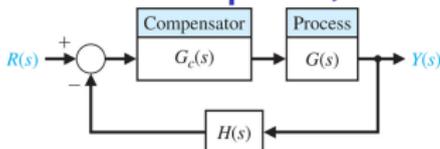
Nyquist's Stability Criterion as a Design Tool

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Relationship to transient response

## Example with open loop RHP pole, PD control

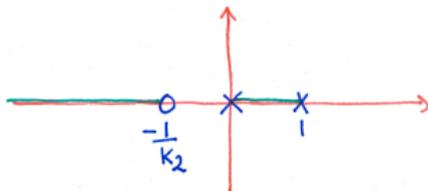


- $G(s) = \frac{1}{s(s-1)}$  and  $H(s) = 1$ .  $L(s) = G_C(s)G(s)$ .
- In the VTOL aircraft example, showed that closed-loop can be stabilized by lead compensation,  $G_C(s) = \frac{K_C(s+z)}{(s+p)}$
- It can also be stabilized by PD comp.,  $G_C(s) = K_1(1 + K_2s)$ . (Under the presumption that this can be realized. It can be realized when we have “velocity” feedback.)
- Using the root locus, we can show that when  $K_2 > 0$  there is a  $K_1 > 0$  that stabilizes the closed loop (see next page)
- Can we see that in the Nyquist diagram?
- Plot the Nyquist diagram of  $L(s) = G_C(s)G(s)$ , where  $G(s) = \frac{K_1}{s(s-1)}$  and  $G_C(s) = 1 + K_2s$

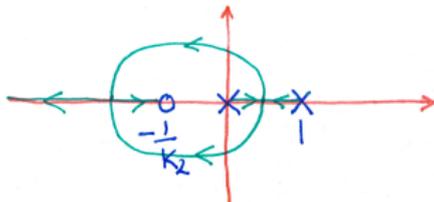
## Root locus analysis

Root locus of  $(1 + K_2 s) \frac{1}{s(s-1)}$  for a given  $K_2 > 0$

- Poles, zero and active sections of real axis



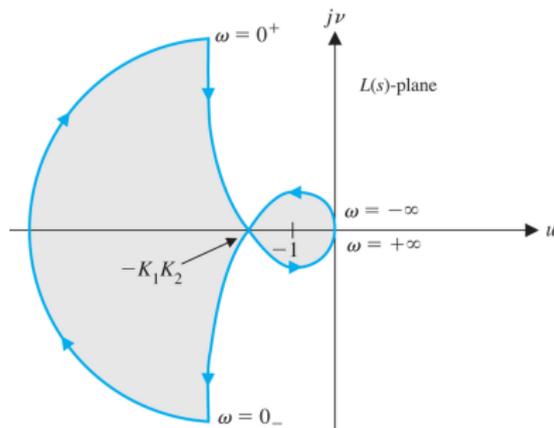
- Complete root locus



Conclusion: For any given  $K_2 > 0$  there is a  $\bar{K}_1 > 0$  such that closed loop is stable for all  $K_1 > \bar{K}_1$ . We can find  $\bar{K}_1$  using Routh-Hurwitz

## Nyquist diagram of

$$(1 + K_2 s) \frac{K_1}{s(s-1)}$$



- Recall that  $P_L = 1$
- If  $K_1 K_2 > 1$ , there is one anti-clockwise encirc. of  $-1$
- In that case, number closed-loop poles in RHP is  $-1 + 1 = 0$  and the closed loop is stable
- Consistent with root locus analysis; but gives  $\bar{K}_1 = 1/K_2$  directly

## One more example

$$L(s) = \frac{K(s-2)}{(s+1)^2}$$

Open loop is stable, but has non-minimum phase (RHP) zero

$$L(j\omega) = \frac{K\sqrt{\omega^2+4}}{\omega^2+1} \angle (180^\circ - \text{atan}(\omega/2) - 2\text{atan}(\omega))$$

- For small positive  $\omega$ ,  $L(j\omega) \approx 2K \angle 180^\circ$
- For large positive  $\omega$ ,  $L(j\omega) \approx \frac{K}{\omega} \angle -90^\circ$
- In between, phase decreases monotonically,  $180^\circ \rightarrow -90^\circ$ .  
magnitude decreases monotonically (Bode mag dia.)
- $L(j\omega) = \frac{2K(2\omega^2-1+j\omega(5-\omega^2))}{(1+\omega^2)^2}$ ; When  $\omega = \sqrt{5}$ ,  $L(j\omega) = K/2$
- When  $s = re^{j\theta}$  with  $r \rightarrow \infty$  and  $\theta : 90^\circ \rightarrow -90^\circ$ ,  
 $L(s) \rightarrow (K/r)e^{-j\theta}$

# Nyquist plot of $L(s)/K$

Tim Davidson

Transfer functions

Frequency Response

Plotting the freq. resp.

Mapping Contours

Nyquist's criterion

Ex: servo, P control

Ex: unst., P control

Ex: unst., PD contr.

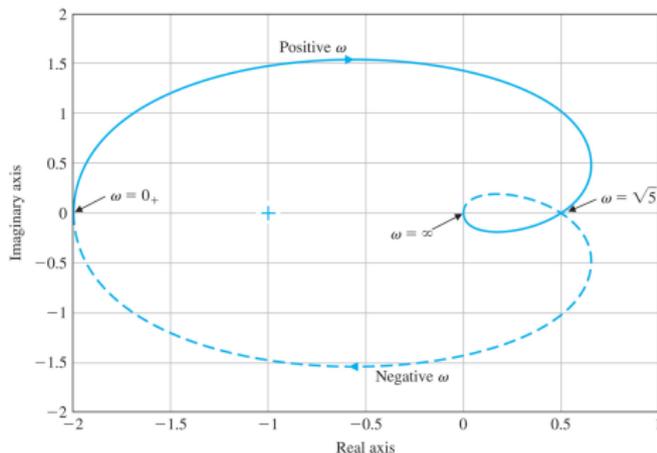
Ex: RHP Z, P contr.

Nyquist's Stability Criterion as a Design Tool

Relative Stability

Gain margin and Phase margin

Relationship to transient response



- Number of open loop RHP poles: 0
- Number of clockwise encirclements of  $-1$ :  
if  $K < 1/2$ : 0;      if  $K > 1/2$ : 1
- Hence closed loop is  
stable for  $K < 1/2$ ;      unstable for  $K > 1/2$
- This is what we would expect from root locus

$$\text{Root locus of } L(s) = \frac{s-2}{(s+1)^2}$$

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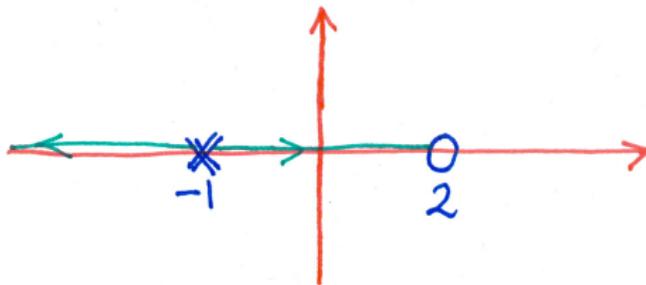
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Nyquist's Stability Criterion as a Design Tool

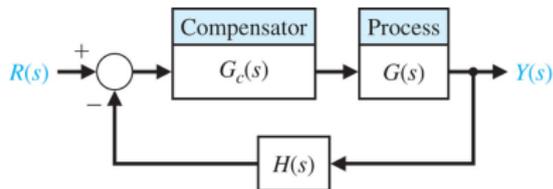
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# Nyquist's Criterion (Review)



- Consider a unity feedback system with an open loop transfer function  $L(s) = G_c(s)G(s)H(s)$ ,
- Let  $P_L$  denote the number of poles of  $L(s)$  in open RHP
- Consider the modified Nyquist Contour in the  $s$ -plane (looping to the right of any poles or zeros on the  $j\omega$ -axis)
- Let  $\Gamma_L$  denote image of mod. Nyquist Contour under  $L(s)$
- Let  $N_L$  denote the number of clockwise encirclements that  $\Gamma_L$  makes of the point  $(-1, 0)$
- **Nyquist's Stability Criterion:**

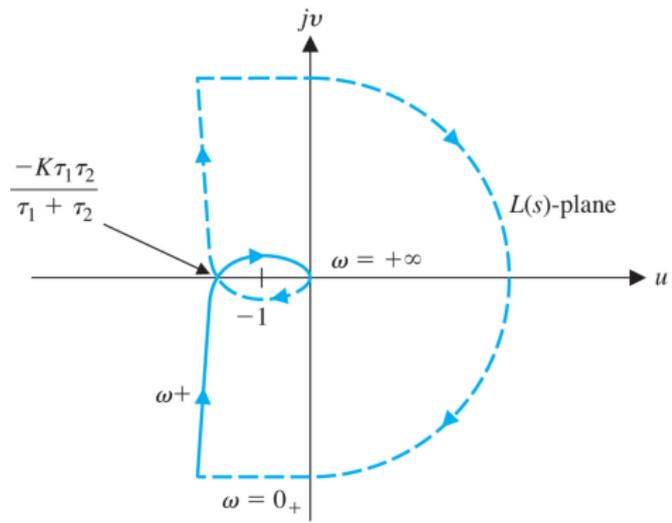
$$\text{Number of closed-loop poles in open RHP} = N_L + P_L$$

# Relative Stability: Introductory Example

Consider

$$L(s) = \frac{K}{s(\tau_1 s + 1)(\tau_2 s + 1)}$$

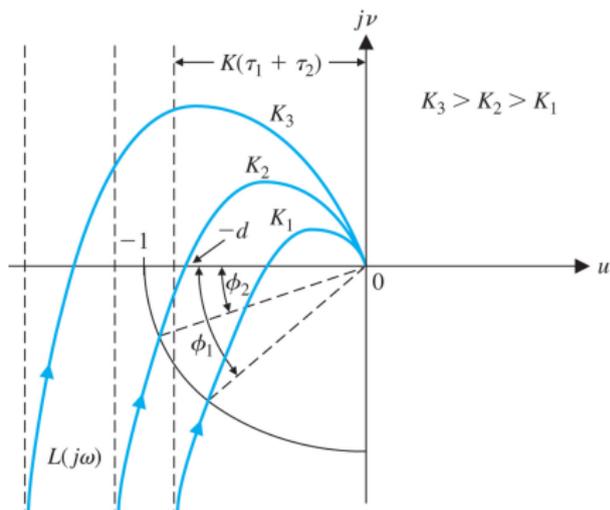
Nyquist Diagram:



- Transfer functions
- Frequency Response
- Plotting the freq. resp.
- Mapping Contours
- Nyquist's criterion
  - Ex: servo, P control
  - Ex: unst., P control
  - Ex: unst., PD contr.
  - Ex: RHP Z, P contr.
- Nyquist's Stability Criterion as a Design Tool
- Relative Stability
  - Gain margin and Phase margin
  - Relationship to transient response

## Zoom in

Since  $L(s)$  is minimum phase (no RHP zeros), we can zoom in



For a given  $K$ ,

- how much extra gain would result in instability?  
we will call this the gain margin
- how much extra phase lag would result in instability?  
we will call this the phase margin

## Formal definitions

- **Gain margin:**  $\frac{1}{|L(j\omega_x)|}$ ,  
where  $\omega_x$  is the frequency at which  $\angle L(j\omega)$  reaches  $-180^\circ$   
amplifying the open-loop transfer function by this amount  
would result in a marginally stable closed loop
- **Phase margin:**  $180^\circ + \angle L(j\omega_c)$ ,  
where  $\omega_c$  is the frequency at which  $|L(j\omega)|$  equals 1  
adding this much phase lag would result in a marginally  
stable closed loop
- These margins can be read from the Bode diagram

Tim Davidson

Transfer functions

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Nyquist's criterion

Ex: servo, P control  
Ex: unst., P control  
Ex: unst., PD contr.  
Ex: RHP Z, P contr.

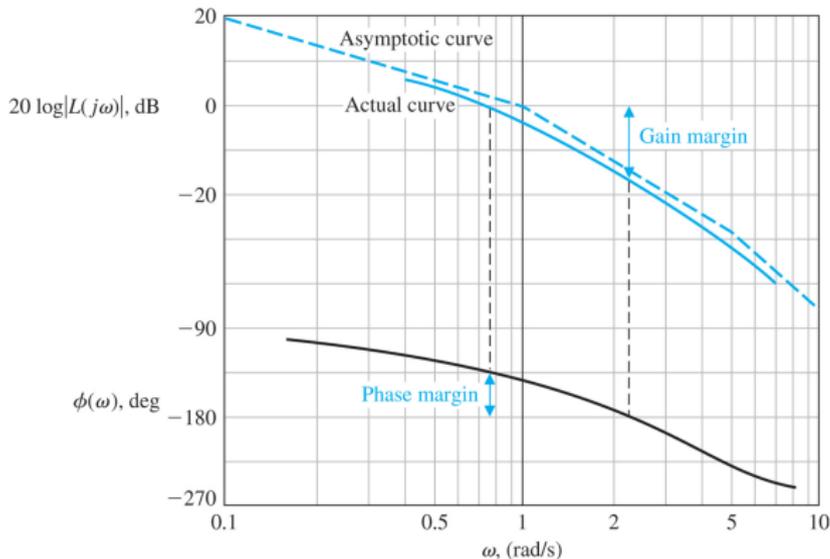
Nyquist's Stability Criterion as a Design Tool

Relative Stability

Gain margin and Phase margin

Relationship to transient response

# Bode diagram



$$L(j\omega) = \frac{1}{j\omega(1 + j\omega)(1 + j\omega/5)}$$

- Gain margin  $\approx 15$  dB
- Phase margin  $\approx 43^\circ$

## Phase margin and damping

Tim Davidson

Transfer  
functionsFrequency  
ResponsePlotting the  
freq. resp.Mapping  
ContoursNyquist's  
criterion

Ex: servo, P control

Ex: unst., P control

Ex: unst., PD contr.

Ex: RHP Z, P contr.

Nyquist's  
Stability  
Criterion as a  
Design Tool

Relative Stability

Gain margin and  
Phase marginRelationship to  
transient response

- Consider a second-order open loop of the form

$$L(s) = \frac{\omega_n^2}{s(s+2\zeta\omega_n)}, \text{ with } \zeta < 1$$

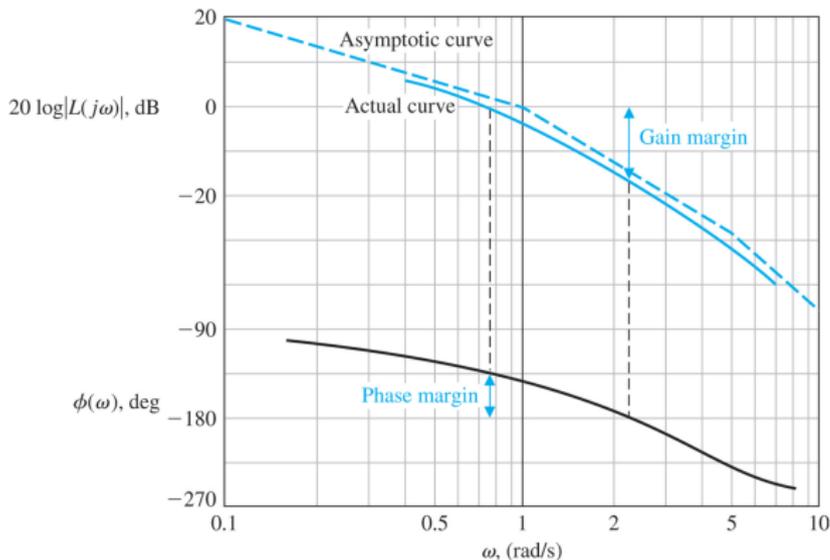
- Closed-loop poles  $s_1, s_2 = -\zeta\omega_n \pm j\omega_n\sqrt{1-\zeta^2}$
- Let  $\omega_c$  be the frequency at which  $|L(j\omega)| = 1$
- Square and rearrange:  $\omega_c^4 + 4\zeta^2\omega_n^2\omega_c^2 - \omega_n^4 = 0$ ;  
Equivalently,  $\frac{\omega_c^2}{\omega_n^2} = \sqrt{4\zeta^4 + 1} - 2\zeta^2$
- By definition,  $\phi_{pm} = 180^\circ + \angle L(j\omega_c)$

- Hence

$$\phi_{pm} = \text{atan}\left(\frac{2}{\sqrt{(4 + 1/\zeta^4)^{1/2} - 2}}\right)$$

- Phase margin is an explicit function of damping ratio!
- Approximation: for  $\zeta < 0.7$ ,  $\zeta \approx 0.01\phi_{pm}$ , where  $\phi_{pm}$  is measured in degrees

## Previous example



$$L(j\omega) = \frac{1}{j\omega(1 + j\omega)(1 + j\omega/5)}$$

- Phase margin  $\approx 43^\circ$
- Damping ratio  $\approx 0.43$