ELEC ENG 4CL4 – Control System Design

Solutions to Homework Assignment #1

1. Consider a system that obeys the differential equation:

$$\frac{d^2x}{dt^2} + 2\frac{dx}{dt} + \cos x = 0.$$

- a. Linearize this equation around the operating point $x = \pi/4$.
- b. Derive a state-space representation of the linear equation found in part a. (25 pts)
- a. The nonlinear term $\cos x$ is linearized via the Taylor series approximation:

$$f(x) = \cos x \approx \cos\left(\frac{\pi}{4}\right) + \frac{df}{dx}\Big|_{x=\frac{\pi}{4}} \left(x - \frac{\pi}{4}\right) = \cos\left(\frac{\pi}{4}\right) - \sin\left(\frac{\pi}{4}\right) \left(x - \frac{\pi}{4}\right)$$
$$= \frac{1}{\sqrt{2}} + \frac{\pi}{4\sqrt{2}} - \frac{x}{\sqrt{2}},$$

giving the linear equation:

$$\frac{d^2x}{dt^2} + 2\frac{dx}{dt} - \frac{x}{\sqrt{2}} = -\frac{1}{\sqrt{2}} - \frac{\pi}{4\sqrt{2}}.$$

Note that an equivalent model in terms of $\Delta x = x - \frac{\pi}{4}$ is:

$$\frac{d^2\Delta x}{dt^2} + 2\frac{d\Delta x}{dt} - \frac{\Delta x}{\sqrt{2}} = -\frac{1}{\sqrt{2}}.$$

b. To obtain a state-space representation of the linearized model, we define two state variables $x_1 = x$ and $x_2 = \frac{dx}{dt}$, the input variable $u = \frac{1}{\sqrt{2}} + \frac{\pi}{4\sqrt{2}}$, and the output variable y = x, giving:

$$\frac{dx_1}{dt} = x_2,$$

$$\frac{dx_2}{dt} = \frac{x_1}{\sqrt{2}} - 2x_2 - u,$$

$$y = x_1.$$

Note that an equivalent model in terms of $\Delta x = x - \frac{\pi}{4}$ and $\Delta u = \frac{1}{\sqrt{2}}$ is:

$$\begin{split} \frac{d\Delta x_1}{dt} &= \Delta x_2, \\ \frac{d\Delta x_2}{dt} &= \frac{\Delta x_1}{\sqrt{2}} - 2\Delta x_2 - \Delta u, \\ \Delta y &= \Delta x_1. \end{split}$$

2. A two-phase (i.e., two-input) permanent magnet stepper motor can be described by the following set of differential equations:

$$\begin{split} \frac{d^2\theta}{dt^2} &= -K_2 i_a \sin\left(K_3\theta\right) + K_2 i_b \cos\left(K_3\theta\right) - K_1 \frac{d\theta}{dt}, \\ \frac{di_a}{dt} &= -K_5 i_a + K_4 \frac{d\theta}{dt} \sin\left(K_3\theta\right) + K_6 v_a, \\ \frac{di_b}{dt} &= -K_5 i_b - K_4 \frac{d\theta}{dt} \cos\left(K_3\theta\right) + K_6 v_b, \end{split}$$

where θ is angular displacement of the rotor, i_a and i_b are the currents in the two phases, v_a and v_b are the voltages applied the two phases (i.e., the inputs), and K_1, \dots, K_6 are constants.

- a. Derive a state-space representation of this system.
- b. Linearize the state-space model found in part a around the operating point $\theta = \text{constant}$. (25 pts)
- a. We assign four state variables $x_1 = \theta$, $x_2 = \frac{d\theta}{dt}$, $x_3 = i_a$ and $x_4 = i_b$, two input variables $u_1 = v_a$ and $u_2 = v_b$, and one output variable $y = \theta$, giving the state-space representation:

$$\frac{dx_1}{dt} = f_1(x_1, x_2, x_3, x_4, u_1, u_2) = x_2$$

$$\frac{dx_2}{dt} = f_2(x_1, x_2, x_3, x_4, u_1, u_2) = -K_2 x_3 \sin(K_3 x_1) + K_2 x_4 \cos(K_3 x_1) - K_1 x_2,$$

$$\frac{dx_3}{dt} = f_3(x_1, x_2, x_3, x_4, u_1, u_2) = -K_5 x_3 + K_4 x_2 \sin(K_3 x_1) + K_6 u_1,$$

$$\frac{dx_4}{dt} = f_4(x_1, x_2, x_3, x_4, u_1, u_2) = -K_5 x_4 - K_4 x_2 \cos(K_3 x_1) + K_6 u_2,$$

$$y = g(x_1, x_2, x_3, x_4, u_1, u_2) = x_1.$$

b. To simplify the linearization of this state-space model, we reformulate it in terms of the distances from the operating point $(x_{1Q}, x_{2Q}, x_{3Q}, x_{4Q}, u_{1Q}, u_{2Q}, y_{Q})$:

$$\Delta x_1 = x_1 - x_{1Q},$$

$$\Delta x_2 = x_2 - x_{2Q},$$

$$\Delta x_3 = x_3 - x_{3Q},$$

$$\Delta x_4 = x_4 - x_{4Q},$$

$$\Delta u_1 = u_1 - u_{1Q},$$

$$\Delta u_2 = u_2 - u_{2Q},$$

$$\Delta y = y - y_{Q}.$$

Defining the vectors:

$$\Delta \mathbf{x} \triangleq \begin{bmatrix} \Delta x_1 & \Delta x_2 & \Delta x_3 & \Delta x_4 \end{bmatrix}^T$$
, $\Delta \dot{\mathbf{x}} \triangleq \begin{bmatrix} \Delta \dot{x}_1 & \Delta \dot{x}_2 & \Delta \dot{x}_3 & \Delta \dot{x}_4 \end{bmatrix}^T$ and $\Delta \mathbf{u} \triangleq \begin{bmatrix} \Delta u_1 & \Delta u_2 \end{bmatrix}^T$,

The linearized state-space model is obtained from the Taylor series approximation in the matrix formulation:

$$\begin{split} \mathbf{\Delta}\dot{\mathbf{x}} &= \begin{bmatrix} \frac{\partial f_1}{\partial x_1} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_1}{\partial x_2} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_1}{\partial x_3} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_2}{\partial x_1} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_2}{\partial x_2} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_2}{\partial x_3} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_2}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_3}{\partial x_1} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_3}{\partial x_2} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_3}{\partial x_3} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_3}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_3}{\partial x_1} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_3}{\partial x_2} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_3}{\partial x_3} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_3}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_4}{\partial x_1} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_2} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_3} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_4}{\partial x_1} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_2} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_3} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_2} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_2} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_2} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} \\ \frac{\partial f_4}{\partial x_4} |_{\mathbf{x} = \mathbf{x}_Q} & \frac{\partial f_4}{$$

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3. Given the following differential equation, solve for y(t) using the Laplace transform if all initial conditions are zero:

$$\frac{d^2y}{dt^2} + 12\frac{dy}{dt} + 32y = 32\mu(t),$$

where $\mu(t)$ is the unit step function.

(25 pts)

The Laplace transform of this equation is:

$$s^{2}Y(s) + 12sY(s) + 32Y(s) = 32\frac{1}{s}$$

$$\Rightarrow Y(s)\{s^{2} + 12s + 32\} = \frac{32}{s}.$$

Solving for Y(s) gives:

$$Y(s) = \frac{32}{s(s^2 + 12s + 32)}$$
$$= \frac{32}{s(s+4)(s+8)}$$
$$= \frac{1}{s} - \frac{2}{s+4} + \frac{1}{s+8}.$$

Taking the inverse Laplace transform of Y(s) we obtain:

$$y(t) = 1 - 2e^{-4t} + e^{-8t}, \quad t \ge 0.$$

4. A system has the transfer function:

$$H(s) = \frac{10}{(s+7)(s+8)}.$$

- a. Compute the system's response to the Dirac delta function (unit impulse) $\delta_{_{
 m D}}(t)$.
- b. Compute the system's response to the unit step function $\mu(t)$. (25 pts)
- a. Because the Laplace transform of $\delta_D(t)$ is 1, the impulse response of a system is the inverse Laplace transform of its transfer function:

$$h(t) = \mathcal{L}^{-1} \{ H(s) \} = \mathcal{L}^{-1} \left\{ \frac{10}{(s+7)(s+8)} \right\} = \mathcal{L}^{-1} \left\{ \frac{10}{s+7} - \frac{10}{s+8} \right\}$$
$$= 10e^{-7t} - 10e^{-8t}, \quad t \ge 0.$$

b. The step response of this system is:

$$y(t) = \mathcal{L}^{-1}\left\{H(s)U(s)\right\} = \mathcal{L}^{-1}\left\{\frac{10}{(s+7)(s+8)}\frac{1}{s}\right\} = \mathcal{L}^{-1}\left\{\frac{\frac{10}{56}}{s} - \frac{\frac{80}{56}}{s+7} + \frac{\frac{5}{4}}{s+8}\right\}$$
$$= \frac{10}{56} - \frac{80}{56}e^{-7t} + \frac{5}{4}e^{-8t}, \quad t \ge 0.$$